

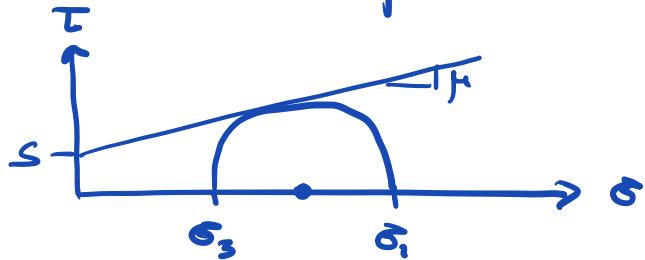
$$(\underline{\mathbf{v}} \cdot \nabla) \underline{\mathbf{v}} \doteq (\nabla \underline{\mathbf{v}}) \underline{\mathbf{v}}$$

Lecture 10: Deformation

Logistics: - HW 4 is due Thursday

- Office hrs tomorrow 3pm

Last time: Mohr circle failure



- Simple states of stress:
hydrostatic, simple & pure shear
plane stress

→ Done with Stress

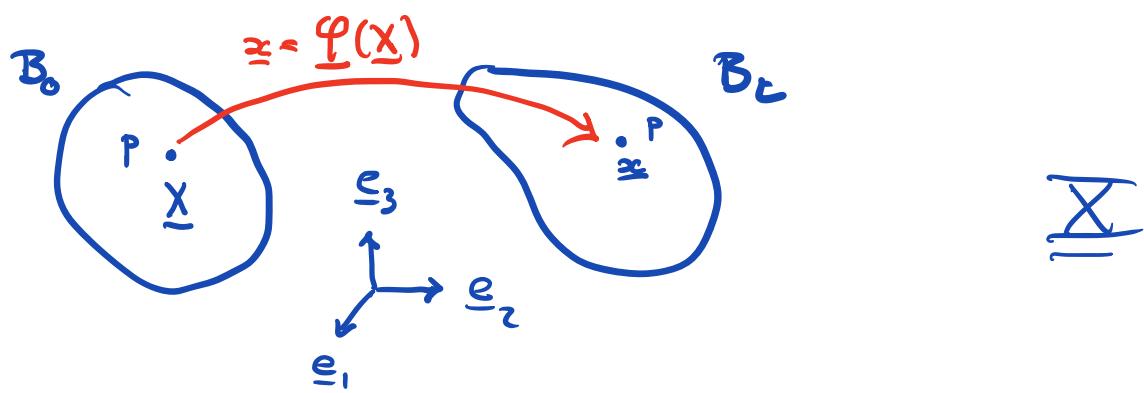
Today: Move to Kinematics

- Deformation mapping
- Measures of Strain
- Deformation gradient

Kinematics

Study of geometry of motion without considering mass and stress
→ quantify strain and rate of strain

Deformation Mapping



B_0 = body in reference config.
initial, undeformed, material

B_t = body in current, spatial, deformed config

P = material point in body

\underline{x} = location of P in B_0

\underline{z} = location of P in B_t

$\varphi(\underline{x})$ = deformation mapping

$\{\underline{e}_1, \underline{e}_2, \underline{e}_3\}$ = frame

$\underline{x} = x_I \underline{e}_I$ x_I = components of \underline{x} in $\{\underline{e}_I\}$

$\underline{x} = x_i \underline{e}_i$ x_i = components of \underline{x} in $\{\underline{e}_i\}$

Convention:

Upper case quantities & indices \rightarrow ref. conf B_0

Lower case quantities & " \rightarrow current conf B_t

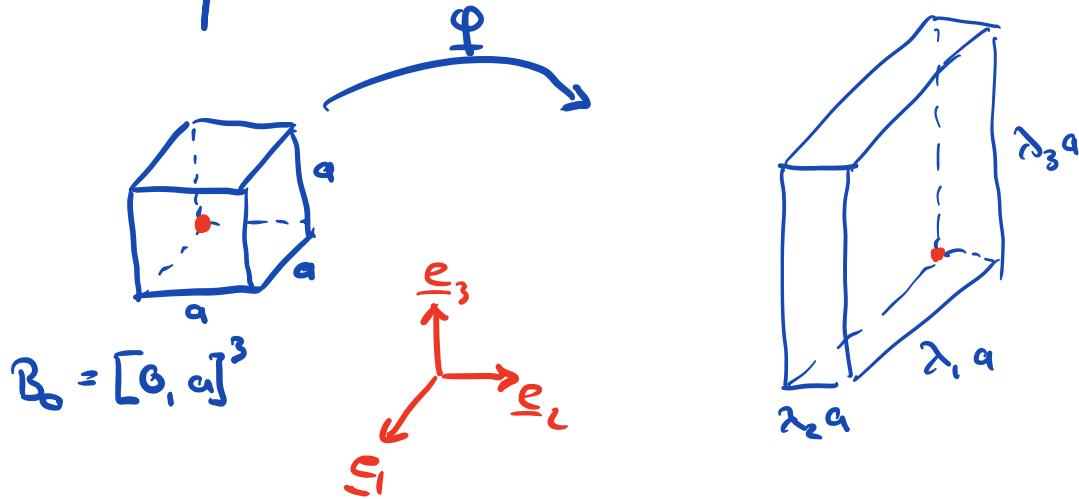
Definition of deformation mapping

$$\underline{x} = \varphi(\underline{X}) = \varphi_i(\underline{X}) \underline{e}_i$$

Displacement of material particle

$$\underline{u}(\underline{x}) = \underline{x} - \underline{X} = \varphi(\underline{x}) - \underline{X}$$

Example: Strecke unit cube



deformation map: $x_1 = \lambda_1 X_1 + v_1$

$$x_2 = \lambda_2 X_2 + v_2$$

$$x_3 = \lambda_3 X_3 + v_3$$

λ : stretch ratios

$$v_i \text{ translation} = 0 \quad \underline{\underline{\lambda}} = \begin{bmatrix} \lambda_1 & \lambda_2 & \lambda_3 \end{bmatrix}$$

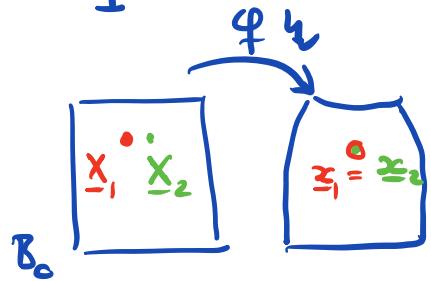
$$\underline{x} = \varphi(\underline{X}) = \lambda_1 X_1 e_1 + \lambda_2 X_2 e_2 + \lambda_3 X_3 e_3$$

$$= \sum_{i=1}^3 X_i e_i = \underline{\underline{\lambda}} \underline{X}$$

Admissible deformations

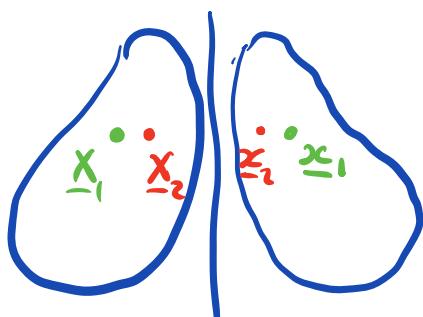
For φ to represent a physically feasible deformation it must satisfy two following conditions:

1) $\varphi: B_0 \rightarrow B_t$ has to be "one to one and onto"



Two separate material points in B_0 cannot be mapped to same point in B_t .

2) $\det(\nabla \varphi) > 0$



The orientation of a body has to be preserved i.e. no reflections.

$Q =$ is ~~an~~ orthogonal matrix

$$\det(R \underline{u}) =$$

$$\underbrace{\det(R)}_{\lambda_1 \lambda_2 \lambda_3} \det(\underline{u})$$

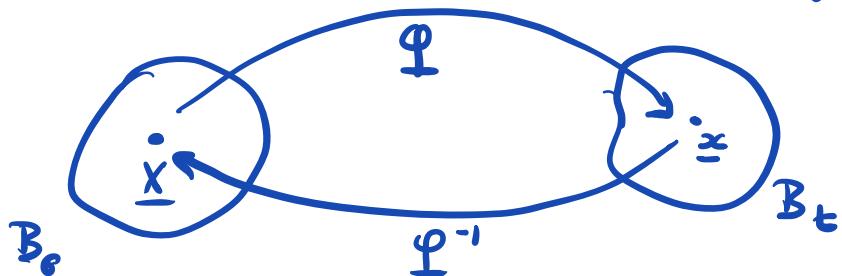
$$\underbrace{\lambda_1 \lambda_2 \lambda_3}_{\geq 0}$$

$\det(Q) > 1$ rotation

$\det(Q) < 1$ reflection

Inverse Mapping

If φ is admissible \Rightarrow well defined inverse φ^{-1}



inverse def. map: $\underline{x} = \underline{\varphi}^{-1}(\underline{z})$

Measures of Strain

In 1D we have simple measures

original:  $\Delta L = \underline{B}_e l - L$

deformed: 

engineering strain: $e = \frac{\Delta L}{L}$ "relative def."

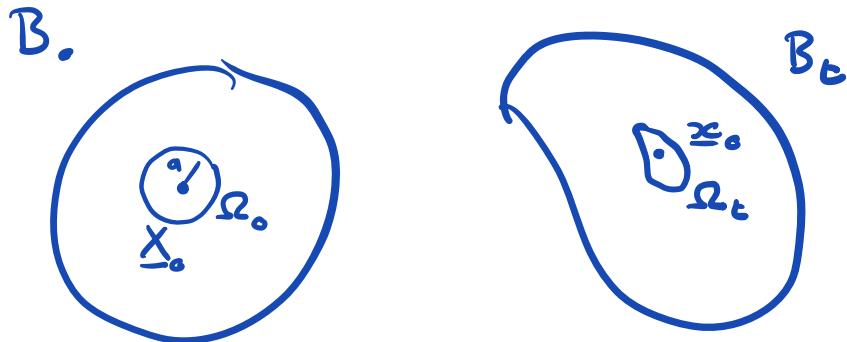
stretch ratio : $\lambda = \frac{l}{L}$ $e = \lambda - 1$

Hencky strain: $\varepsilon = \ln(\lambda)$

Green strain: $\epsilon = \frac{1}{2}(\lambda^2 - 1)$

\Rightarrow Quantification of strain is not unique

We are looking for general 3D approach
that is not limited to small deformations.



Sphere of radius a around X_0 , is
mapped to $\Omega_t = \varphi(\Omega_0)$
 $\Omega_t = \{ \underline{x} \in B_t \mid \underline{x} = \varphi(\underline{x}_0), \underline{x}_0 \in \Omega_0 \}$

Def: The strain at X_0 is any relative
difference between Ω_0 and Ω_t
in limit of $a \rightarrow 0$.

Deformation Gradient

Natural way to quantify strains near \underline{x}_0 .

$$\boxed{\underline{F}(\underline{x}) = \nabla \varphi(\underline{x})} \quad F_{ij} = \frac{\partial \varphi_i}{\partial x_j}$$

Approximate φ using Taylor series around \underline{x}_0 .

$$\begin{aligned}\varphi(\underline{x}) &= \varphi(\underline{x}_0) + \nabla \varphi(\underline{x}_0) (\underline{x} - \underline{x}_0) + O(|\underline{x} - \underline{x}_0|^2) \\ &= \underbrace{\varphi(\underline{x}_0) - \nabla \varphi(\underline{x}_0) \underline{x}_0}_{c} + \nabla \varphi(\underline{x}_0) \underline{x} \\ &= c + \underline{F}(\underline{x}_0) \underline{x}\end{aligned}$$

In vicinity of \underline{x} we have lin. approx

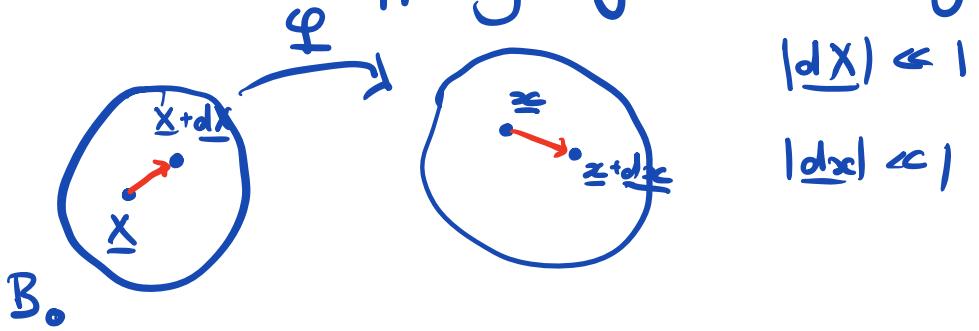
$$\boxed{\varphi(\underline{x}) = c + \underline{F}(\underline{x}_0) \underline{x}}$$

$\Rightarrow \underline{F}(\underline{x})$ describes local deformation

Homogeneous def. \underline{F} is const.

$$\underline{x} = \varphi(\underline{x}) = c + \underline{F} \underline{x} \quad \text{affine def.}$$

Consider mapping of a line segment



$$\underline{x} + \underline{dx} = \underline{\varphi}(\underline{x} + \underline{d\underline{x}}) \approx \underline{\varphi}(\underline{x}) + \underline{\nabla \varphi(\underline{x})} \underline{d\underline{x}}$$

~~\underline{x}~~ ~~$\underline{\nabla \varphi(\underline{x})}$~~

$$\underline{dx} = \underline{F} \underline{d\underline{x}}$$

$$dx_i = F_{ij} d\underline{x}_j$$

(new def.)

\underline{F} maps material vectors into spatial vectors.

Example : Shear deformation



$$\frac{\partial \varphi_1}{\partial x_2}$$

$$\nabla \varphi = \begin{bmatrix} 1 & 2\alpha x_2 \\ 0 & 1 \end{bmatrix} = \underline{\underline{F}}(\underline{x})$$

$$\underline{\underline{F}}(0)\underline{\epsilon}_1 = [1, 0]^T \quad \underline{\epsilon}_1 \text{ is unchanged}$$

$$\underline{\underline{F}}\underline{\epsilon}_2 = [2\alpha x_2, 1]^T$$

In admissible deformations

