

Lecture 19: Lagrangian balance laws

Logistics: —

Last time: - Continuum thermo

- 1st law: $\rho \dot{\phi} = \underline{\underline{\sigma}} : \underline{\underline{d}} - \nabla_x \cdot \underline{q} + \rho r$

$$\frac{\partial}{\partial t}(\rho \phi) + \nabla \cdot (\rho \phi \underline{v} + \underline{q}) = \underline{\underline{\sigma}} : \underline{\underline{d}} - \rho r$$

- 2nd law: $\rho \dot{s} \geq \frac{\rho r}{\theta} - \nabla_x \cdot \left(\frac{\underline{q}}{\theta} \right) \quad \text{C-D}$

⇒ show dir. of heat conduction

- Helmholtz free energy: $\psi = \phi - \theta s$

⇒ $\rho \dot{\psi} \leq \underline{\underline{\sigma}} : \underline{\underline{d}}$ if $\theta = \text{const.}$

stress power is rate of change of free energy.

⇒ ψ is part of ϕ available to do work

- Eulerian form: 21 unknown & 11 eqns!

Today: - Lagrangian balance laws

$$\begin{array}{ccc} \underline{\underline{x}} & \rightarrow & \underline{X} \\ E & & L \end{array}$$

Balance of mass

already showed this

$$\rho_m(\underline{X}) \det \underline{F}(\underline{X}, t) = \rho_0(\underline{X})$$

if φ is known $\rightarrow \underline{F} = \nabla \varphi \rightarrow \rho_m$ is known
mass density is a known quantity

Balance of lin. momentum

Integral balance law

$$\frac{d}{dt} \underline{L}[\Omega_t] = \underline{r}[\Omega_t]$$

$$\text{where } \underline{L}[\Omega_t] = \int_{\Omega_t} \rho(\underline{x}, t) \underline{v}(\underline{x}, t) dV_x$$

$$\underline{r}[\Omega_t] = \int_{\Omega_t} \rho(\underline{x}, t) \underline{b}(\underline{x}, t) dV_x + \int_{\partial \Omega_t} \underline{\sigma} \underline{n} dA_x$$

change variables $\underline{x} \rightarrow \underline{X}$

$$\underline{L}[\Omega_t] = \int_{\Omega_0} \rho_m(\underline{X}, t) \underbrace{\underline{v}_m(\underline{X}, t)}_{\underline{v}(\underline{X}, t) = \dot{\underline{\varphi}}} \underbrace{\det \underline{F}(\underline{X}, t)}_{\underline{J}} dV_X$$

$$\underline{L}[\Omega_t] = \int_{\Omega_0} \rho_0(\underline{X}) \dot{\underline{\varphi}}(\underline{X}, t) dV_X$$

To change surface integral Nausew's form.

$$\underline{n} dA_x = J \underline{F}^{-T} \underline{N} dA_x$$

$$\Gamma[\Omega_t] = \int_{\partial\Omega_t} \underline{\underline{\sigma}} \underline{n} dA_x + \int_{\Omega_t} \rho \underline{b} dV_x$$

$$= \int_{\partial\Omega_0} \underline{\underline{\sigma}}_m \det(\underline{F}) \underline{F}^{-T} \underline{N} dA_x + \int_{\Omega_0} \rho_m \underline{b}_m J dV_x + \int_{\Omega_0} \rho_0 \underline{b}_m dV_x$$

Introduce first Piola-Kirchhoff stress tensor

$$\underline{\underline{P}}(\underline{X}, t) = \det \underline{F} \underline{\underline{\sigma}}_m \underline{F}^{-T}$$

using $\underline{\underline{P}}$ we have

$$\Gamma[\Omega_t] = \int_{\partial\Omega_0} \underline{\underline{P}} \underline{N} dA_x + \int_{\Omega_0} \rho_0 \underline{b}_m dV_x$$

hence we have

$$\frac{d}{dt} \int_{\Omega_0} \rho_0 \dot{\varphi} dV_x = \int_{\partial\Omega_0} \underline{\underline{P}} \underline{N} dA_x + \int_{\Omega_0} \rho_0 \underline{b}_m dV_x$$

$$\int_{\Omega} \rho_0 \ddot{\varphi} dV_x = \int_{\partial\Omega} \nabla_x \cdot \underline{\underline{P}} + \rho_0 \underline{b}_m dV_x$$

hence

$$\rho_0 \ddot{\underline{\varphi}} = \nabla_{\underline{x}} \cdot \underline{\underline{P}} + \rho_0 \underline{b}_m \quad \text{local Lagrangian lin. moment. balance}$$

Note: $\underline{\underline{P}}$ is the natural stress tensor in the material description. It relates the traction to normal of surface

spatial:

$$\underline{t}(\underline{x}, t) = \underline{\underline{S}}(\underline{x}, t) \underline{n}(\underline{x}, t)$$
$$\underline{T}(\underline{X}, t) = \underline{\underline{P}}(\underline{X}, t) \underline{N}(\underline{X}, t)$$

Here \underline{T} is the (nominal) Piola-Kirchhoff traction vector and \underline{t} (true) Cauchy traction vector.

$$d\underline{f} = \underline{t} dA_x = \underline{T} dA_X$$

$\Rightarrow \underline{t} \parallel \underline{T}$ same direction different magn.

Balance of angular momentum

from Eulerian form: $\underline{\underline{\sigma}} = \underline{\underline{\sigma}}^T \Rightarrow \underline{\underline{\sigma}}_m = \underline{\underline{\sigma}}_m^T$

def. $\underline{\underline{P}} = \int \underline{\underline{\sigma}}_m \underline{\underline{F}}^{-T}$

$$\underline{\underline{\sigma}}_m = \frac{1}{J} \underline{\underline{P}} \underline{\underline{F}}^T = \underline{\underline{\sigma}}_m^T = \left(\frac{1}{J} \underline{\underline{P}} \underline{\underline{F}}^T \right)^T = \frac{1}{J} \underline{\underline{F}} \underline{\underline{P}}^T$$

\Rightarrow $\underline{\underline{P}} \underline{\underline{F}}^T = \underline{\underline{F}} \underline{\underline{P}}^T$ constraint from
angular mom. balance

$\Rightarrow \underline{\underline{P}} \neq \underline{\underline{P}}^T \Rightarrow \underline{\underline{P}}$ has 9 indep. components.

ang. mom. balance motivates second

Piola-Kirchhoff stress tensor

$$\underline{\underline{\Sigma}} = \underline{\underline{P}} \underline{\underline{F}}^T \quad \text{so that} \quad \underline{\underline{\Sigma}} = \underline{\underline{\Sigma}}^T$$

Char. of Net working

$$W[\Omega_t] = \mathcal{P}[\Omega_t] - \frac{d}{dt} \mathcal{K}[\Omega_t]$$

$$\mathcal{K}[\Omega_t] = \int_{\Omega_t} \frac{1}{2} \rho |\underline{v}|^2 dV_x$$

$$\mathcal{P}[\Omega_t] = \int_{\Omega_t} \rho \underline{b} \cdot \underline{v} dV_x + \int_{\partial\Omega} \underline{\underline{\underline{\sigma}}} \underline{v} \cdot \underline{n} dA_x$$

used transpose: $\underline{v} \cdot \underline{\underline{\underline{\sigma}}} \underline{n} = \underline{\underline{\underline{\sigma}}}^T \underline{v} \cdot \underline{n} = \underline{\underline{\underline{\sigma}}} \underline{v} \cdot \underline{n}$

Changing variables:

$$\mathcal{K}[\Omega_t] = \int_{\Omega_0} \frac{1}{2} \rho_0 |\dot{\underline{\varphi}}|^2 dV_x$$

$$\mathcal{P}[\Omega_t] = \int_{\Omega_0} \rho_0 \underline{b}_m \cdot \dot{\underline{\varphi}} dV_x + \int_{\partial\Omega_0} \underline{\underline{\underline{\sigma}}}_m \underline{v}_m \cdot \underline{F}^{-T} \underline{N} dA_x$$

$$+ \int_{\partial\Omega_0} \underline{v}_m \cdot \underline{\underline{\underline{\sigma}}}_m \underline{F}^{-T} \underline{N} dA_x$$

$$+ \int \dot{\underline{\varphi}} \cdot \underbrace{\int \underline{\underline{\underline{\sigma}}}_m \underline{F}^{-T} \underline{N} dA_x}_{\underline{\underline{\underline{P}}}} dV_x$$

$$\mathcal{P}[\Omega_t] = \int_{\Omega_0} \rho_0 \underline{b}_m \cdot \dot{\underline{\varphi}} dV_x + \int_{\partial\Omega_0} \dot{\underline{\varphi}} \cdot \underline{\underline{\underline{P}}} \underline{N} dA_x$$

$$= \int_{\Omega} \rho_0 \underline{b}_m \cdot \dot{\underline{\varphi}} dV_x + \int_{\partial\Omega_0} \underline{\underline{\underline{P}}}^T \dot{\underline{\varphi}} \cdot \underline{N} dA_x$$

use identity: $\nabla_x \cdot (\underline{P}^T \dot{\underline{\phi}}) = (\nabla_x \cdot \underline{P}) \cdot \dot{\underline{\phi}} + \underline{P} : \nabla \dot{\underline{\phi}}$

Lagrangian expression for power.

$$\mathcal{P}[\Omega_t] = \int_{\Omega_0} [\nabla_x \cdot \underline{P} + \rho_0 \underline{b}_m] \cdot \dot{\underline{\phi}} + \underline{P} : \dot{\underline{F}} \, dV_x$$

The rate of change of kinetic energy

$$\frac{d}{dt} \mathcal{K}[\Omega_t] = \int_{\Omega} \frac{1}{2} \rho_0 \frac{d}{dt} |\dot{\underline{\phi}}|^2 \, dV_x = \int \dot{\underline{\phi}} \cdot (\rho_0 \ddot{\underline{\phi}}) \, dV_x$$

use lin. mom. balance $\rho_0 \ddot{\underline{\phi}} = \nabla_x \cdot \underline{P} + \rho_0 \underline{b}_m$

$$\frac{d}{dt} \mathcal{K}[\Omega_t] = \int_{\Omega} [\nabla_x \cdot \underline{P} + \rho_0 \underline{b}_m] \cdot \dot{\underline{\phi}} \, dV_x$$

Hence net working

$$\mathcal{W}[\Omega_t] = \mathcal{P}[\Omega_t] - \frac{d}{dt} \mathcal{K}[\Omega_t] = \int_{\Omega_0} \underline{P} : \dot{\underline{F}} \, dV_x$$

Lagrangian def net working

$$\mathcal{W}[\Omega_t] = \int_{\Omega_0} \underline{P} : \dot{\underline{F}} \, dV_x$$

$\underline{\underline{P}} : \underline{\underline{\dot{F}}}$ is the Lagrangian stress power

$\underline{\underline{G}} : \underline{\underline{\dot{d}}}$ is the Eulerian stress power

$\underline{\underline{P}} : \underline{\underline{\dot{F}}}$ is power per unit vol. B_0

$\underline{\underline{G}} : \underline{\underline{\dot{d}}}$ is power per unit vol. B_t

Lagrangian Energy balance

Integral form of 1st Law of thermo

$$\frac{d}{dt} U[\Omega_t] = Q[\Omega_t] + W[\Omega_t]$$

where

$$U[\Omega_t] = \int_{\Omega_t} \rho u \, dV_x \quad \phi \equiv u$$

$$\int_{\Omega_0} \rho_0 u_m \, dV_x \quad u = u_m$$

$$\int_{\Omega_0} \rho_0 U \, dV_x$$

Rate of net heating

$$Q[\Omega_t] = \int_{\Omega_t} \rho r \, dV_x - \int_{\partial\Omega_t} \mathbf{q} \cdot \mathbf{n} \, dA_x$$

Nanson's formula: $\underline{n} dA_x = J \underline{F}^{-T} \underline{N} dA_x$

$$Q[\Omega_t] = \int_{\Omega_0} \rho_0 R dV_x - \int_{\partial\Omega_0} q_m \cdot J \underline{F}^{-T} \underline{N} dA_x$$

$$R = r_m$$

use transpose: $\underbrace{J \underline{F}^{-1} q_m}_{\underline{Q}} \cdot \underline{N} = q_m \cdot J \underline{F}^{-T} \underline{N}$
Q material heat flux

1st law with heating

$$Q[\Omega_t] = \int_{\Omega_0} \rho_0 R dV_x - \int_{\partial\Omega_0} \underline{Q} \cdot \underline{N} dA_x$$

subst. into 1st law + div. flux + localization

$$\boxed{\rho_0 \dot{U} = \underline{P} : \underline{\dot{F}} - \nabla_x \cdot \underline{Q} + \rho_0 R} \quad \text{local Lagrangian energy balance}$$

Lagrangian form 2nd law

integral balance

$$\int_{\Omega_t} \rho \dot{s} dV_x \geq \int_{\Omega_t} \frac{\rho r}{\Theta} dV_x - \int_{\partial\Omega_t} \frac{q \cdot n}{\Theta} dA_x$$

change variables

$$\int_{\Omega_0} \rho_0 S \, dV_X \geq \int_{\Omega_0} \frac{\rho_0 R}{\Theta} \, dV_X - \int_{\partial\Omega_0} \frac{Q \cdot N}{\Theta} \, dA_X$$

$S = s_m \quad \Theta = \theta_m$

using. div. theorem + localization.

$$\rho_0 \dot{S} \geq \frac{\rho_0 R}{\Theta} - \nabla_X \cdot \frac{Q}{\Theta} \quad \text{Clausius-Duhem}$$

in Lag. form

introduce Helmholtz free energy

$$\Psi(\underline{X}, t) = U(\underline{X}, t) - \Theta(\underline{X}, t) S(\underline{X}, t)$$

similar to Eulerian case

$$\rho_0 \dot{\Psi} \leq \underline{\underline{P}} : \underline{\underline{F}} - \rho_0 S \dot{\Theta} - \frac{1}{\Theta} Q \cdot \nabla_X \Theta$$

reduced C-D inequality

$$\Rightarrow \rho_0 \dot{\Psi} = \underline{\underline{P}} : \underline{\underline{F}}$$

Summary of Lagrangian formulation

Balance laws

Kinematic: $\underline{v} = \dot{\underline{\varphi}}$ 3

lin. mom: $\rho_0 \dot{\underline{v}} = \nabla_x \cdot \underline{P} + \rho_0 b_m$ 3

ang. mom: $\underline{\Sigma} = \underline{\Sigma}^T \quad \underline{P} \underline{F}^T = \underline{F} \underline{P}^T$ 3

energy bal: $\rho_0 \dot{u} = \underline{P} : \dot{\underline{F}} - \nabla_x \cdot \underline{Q} + \rho_0 R$ $\frac{+ 1}{10}$

Lagrangian fields

φ	\underline{v}	\underline{P}	$\underline{\Sigma}$	\underline{Q}	u		
3	3	9	1	3	1	$\stackrel{12}{=}$	$20 - 3 - 1 - 3 - 1$

We have 20 unknowns and 10 equations

\Rightarrow Lag. form. has one less unknown & equ

compared to Eulerian, because ρ is known

Remarks:

1) In many situations \underline{v} is not needed explicitly

\Rightarrow 17 unknowns & 7 eqns

2) Need const. equations that relate

$$\underline{\Sigma} = \underline{P} \underline{F}^T, \underline{Q}, \underline{u} \text{ to } \underline{q} \text{ and } \underline{\Theta}$$

3) If thermal effects are also neglected

\Rightarrow 12 unknowns and 6 eqns

to close system requires 6 eqns

relating $\underline{\Sigma}$ to \underline{q}