

## Lecture 13: Cauchy-Green Strain Tensors

Logistics: - HW 5 due

- HW 4 had issue with fault image
- HW 6 will be posted

Last time: Analysis of local deformation

- Changes in surface area:  $\underline{n} dA_x = J \underline{F}^T \underline{N} dA_x$

- Polar Decomposition:  $\underline{F} = \overset{\text{right}}{\underline{R}} \underline{U} = \overset{\text{left}}{\underline{V}} \underline{R}$   $\det(\underline{F}) \neq 0$

$\underline{R}$  = rotation  $\underline{U} = \sqrt{\underline{F}^T \underline{F}}$   $\underline{V} = \sqrt{\underline{F} \underline{F}^T}$  stretches

- Stretch-rotation decoup.:  $\underline{F} = \underline{r} \circ \underline{s}_1 = \underline{s}_2 \circ \underline{r}$

Today: - <sup>right</sup> Cauchy-Green strain tensor  $\underline{C}$

- Interpretation of  $\underline{C}$
- Cauchy-Green strain relations
- Components of  $\underline{C}$

## Cauchy - Green Strain Tensor

$$\varphi: B \rightarrow B_e \quad \underline{x} = \varphi(\underline{X}) = \underline{c} + \underline{F}\underline{X} \quad \underline{F} = \nabla \varphi$$

(right) Cauchy - Green strain tensor

$$\underline{C} = \underline{F}^T \underline{F} = \underline{U}^2$$

only contains information about stretches

Can get  $\underline{C}$  from  $\underline{F}$  but not the other way

Why not use  $\underline{U}$ ?  $\underline{C}$  is much simpler to work with

Example:

$$\underline{F} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & 2 & 1 \\ 0 & 1 & 2 \end{pmatrix}$$

$$\underline{C} = \underline{F}^T \underline{F} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & 5 & 4 \\ 0 & 4 & 5 \end{pmatrix} \quad \checkmark$$

To get  $\underline{U} = \sqrt{\underline{C}}$   $\Rightarrow$  need to solve eigen value problem:

$$\begin{vmatrix} 1-\mu & 0 & 0 \\ 0 & 5-\mu & 4 \\ 0 & 4 & 5-\mu \end{vmatrix} = (1-\mu)(5-\mu)^2 - 16(1-\mu) = 0$$

$$\mu_{1/2} = 1 \quad \mu_3 = 9$$

$$\text{eigen vectors: } \underline{u}_1 = \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} \quad \underline{u}_2 = \frac{1}{\sqrt{2}} \begin{bmatrix} 2 \\ -1 \\ 1 \end{bmatrix} \quad \underline{u}_3 = \frac{1}{\sqrt{2}} \begin{bmatrix} 0 \\ 1 \\ 1 \end{bmatrix}$$

$$\Rightarrow \underline{U} = \sqrt{\underline{C}} = \sum_{i=1}^3 \sqrt{\mu_i} \underline{u}_i \otimes \underline{u}_i = \begin{pmatrix} 1 & 0 & 0 \\ 0 & 2 & 1 \\ 0 & 1 & 2 \end{pmatrix}$$

Note:  $\underline{F} = \underline{U}$  because there is no rotation.

Here  $\mu$  are eigen values of  $\underline{C}$

$$\underline{U} = \sum_{i=1}^3 \lambda_i \underline{u}_i \otimes \underline{u}_i$$

$\lambda_i$ 's principal stretches

$$\underline{C} = \underline{U}^2 = \sum_{i=1}^3 \lambda_i^2 \underline{u}_i \otimes \underline{u}_i$$

$\mu_i = \lambda_i^2$  eigen values of  $\underline{C}$  are

squares of principal stretches

$$x_i \underline{e}_i = F_{ij} X_j \underline{e}_i \quad \underline{x} = \underline{F} \underline{X}$$

$$C_{KL} = F_{iK} F_{iL} \quad \text{"material strain tensor"}$$

### Other strain tensors

$$\text{I) } \underline{\underline{E}} = \frac{1}{2} (\underline{\underline{C}} - \underline{\underline{I}}) \quad \text{Green-Lagrange strain tensor}$$

$$E_{KL} = \frac{1}{2} (C_{KL} - \delta_{KL}) \quad \text{material tensor}$$

$\Rightarrow$  linear theory

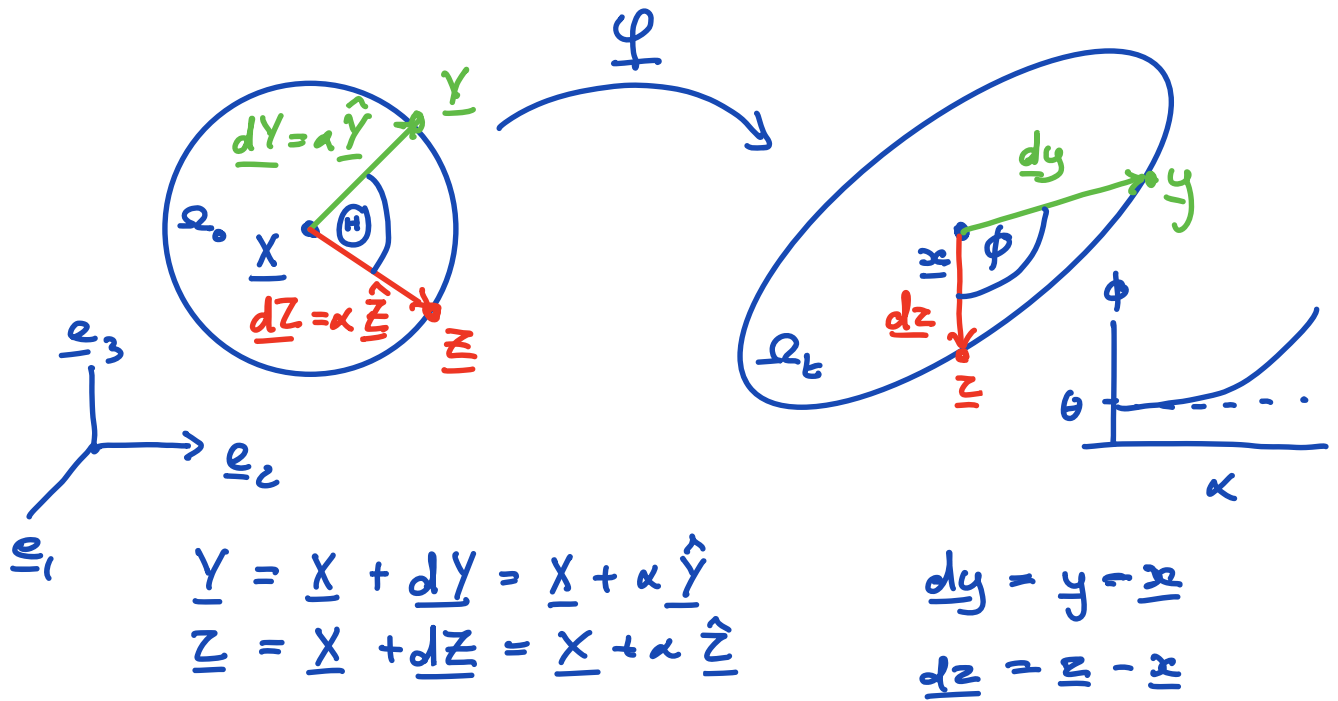
$$\text{II) } \underline{\underline{b}} = \underline{\underline{F}} \underline{\underline{F}}^T = \underline{\underline{V}}^2 \quad \text{left Cauchy Green strain tensor}$$

$$b_{kl} = F_{kI} F_{lI} \quad \text{"spatial tensor"}$$

$$\text{III) } \underline{\underline{e}} = \frac{1}{2} (\underline{\underline{I}} - \underline{\underline{F}}^{-T} \underline{\underline{F}}^{-1}) \quad \text{Euler-Almansi tensor}$$

$$e_{kl} = \frac{1}{2} (\delta_{kl} - F_{Ik}^{-1} F_{Il}^{-1}) \quad \text{"spatial tensor"}$$

## Interpretation of $\underline{C}$



In limit  $\alpha \rightarrow 0$   $\cos \phi \rightarrow \cos \theta(\hat{\underline{Y}}, \hat{\underline{Z}})$

## Cauchy - Green Strain Relations

$$\lambda(\hat{\underline{Y}}) = \sqrt{\hat{\underline{Y}} \cdot \underline{C} \hat{\underline{Y}}}$$

$$\cos \theta(\hat{\underline{Y}}, \hat{\underline{Z}}) = \frac{\hat{\underline{Y}} \cdot \underline{C} \hat{\underline{Z}}}{\sqrt{\hat{\underline{Y}} \cdot \underline{C} \hat{\underline{Y}}} \sqrt{\hat{\underline{Z}} \cdot \underline{C} \hat{\underline{Z}}}}$$

$\Rightarrow$  stretch at  $\underline{x}$  in direction  $\hat{\underline{Y}}$

$\lambda(\hat{\underline{Y}})$     $\lambda(\hat{\underline{Z}})$

## I. Stretched

In the limit  $\alpha \rightarrow 0$

$$\frac{|d\underline{y}|}{|d\underline{Y}|} \rightarrow \lambda(\hat{\underline{Y}}) \quad \text{and} \quad \frac{|d\underline{z}|}{|d\underline{Z}|} \rightarrow \lambda(\hat{\underline{Z}})$$

$$\begin{aligned} |d\underline{y}|^2 &= d\underline{y} \cdot d\underline{y} = (\underline{F} d\underline{Y}) \cdot (\underline{F} d\underline{Y}) = d\underline{Y} \cdot \underbrace{\underline{F}^T \underline{F}}_{\underline{C}} d\underline{Y} \\ &= d\underline{Y} \cdot \underline{C} d\underline{Y} \quad d\underline{Y} = \alpha \hat{\underline{Y}} \\ &= \alpha^2 \hat{\underline{Y}} \cdot \underline{C} \hat{\underline{Y}} \end{aligned}$$

$$|d\underline{Y}|^2 = |\alpha \hat{\underline{Y}}|^2 = \alpha^2$$

$$\frac{|d\underline{y}|}{|d\underline{Y}|} = \sqrt{\hat{\underline{Y}} \cdot \underline{C} \hat{\underline{Y}}} = \lambda(\hat{\underline{Y}})$$

If  $(\lambda_i, \hat{\underline{U}}_i)$  are eigenpairs of  $\underline{U}$

$$(\underline{C} - \lambda_i^2 \underline{I}) \hat{\underline{U}}_i = \underline{0} \quad \text{eigen problem for } \underline{C}$$

$$\underline{C} \hat{\underline{U}}_i - \lambda_i^2 \hat{\underline{U}}_i = \underline{0}$$

$$\hat{\underline{U}}_i \cdot \underline{C} \hat{\underline{U}}_i - \lambda_i^2 \underbrace{\hat{\underline{U}}_i \cdot \hat{\underline{U}}_i}_1 = 0 \quad \lambda(\hat{\underline{Y}}) = \sqrt{\hat{\underline{Y}} \cdot \underline{C} \hat{\underline{Y}}}$$

$$\lambda_i = \sqrt{\hat{\underline{U}}_i \cdot \underline{C} \hat{\underline{U}}_i} \quad \lambda(\hat{\underline{U}}_i) \quad \hat{\underline{Y}} = \hat{\underline{U}}_i$$

⇒ proved that  $\lambda_i$ 's are principal stresses

Use argument similar to max/min normal stresses to show that  $\lambda_i$ 's correspond to the max/min stresses.

## II, Shear

The shear  $\gamma(\hat{\underline{y}}, \hat{\underline{z}})$  at  $\underline{x}$  is the change in angle between the two directions  $\hat{\underline{y}}$  &  $\hat{\underline{z}}$

$$\gamma(\hat{\underline{y}}, \hat{\underline{z}}) = \Theta(\hat{\underline{y}}, \hat{\underline{z}}) - \theta(\hat{\underline{y}}, \hat{\underline{z}})$$

where  $\lim_{\alpha \rightarrow 0} \cos \phi = \cos \theta(\hat{\underline{y}}, \hat{\underline{z}})$

$$d\underline{y} \cdot d\underline{z} = |d\underline{y}| |d\underline{z}| \cos \phi$$

$$\cos \phi = \frac{d\underline{y} \cdot d\underline{z}}{|d\underline{y}| |d\underline{z}|}$$

$$\underline{dy} \cdot \underline{dz} = (\underline{F} \underline{dY}) \cdot (\underline{F} \underline{dZ}) = \underline{dY} \cdot \underline{F}^T \underline{F} \underline{dZ}$$

$$= \underline{dY} \cdot \underline{C} \underline{dZ} = \alpha^2 \hat{\underline{Y}} \cdot \underline{C} \hat{\underline{Z}}$$

$$|d\underline{y}| = \alpha \lambda(\underline{Y}) = \alpha \sqrt{\hat{\underline{Y}} \cdot \underline{C} \hat{\underline{Y}}} \quad |d\underline{z}| = \alpha \sqrt{\hat{\underline{Z}} \cdot \underline{C} \hat{\underline{Z}}}$$

$$\cos \phi = \frac{\hat{\underline{Y}} \cdot \underline{C} \hat{\underline{Z}}}{\sqrt{\hat{\underline{Y}} \cdot \underline{C} \hat{\underline{Y}}} \sqrt{\hat{\underline{Z}} \cdot \underline{C} \hat{\underline{Z}}}} \quad \checkmark \rightarrow \cos \theta(\hat{\underline{Y}}, \hat{\underline{Z}})$$

Components of  $\underline{C}$

For frame  $\{\underline{e}_I\}$

$$C_{II} = \lambda^2(\underline{e}_I)$$

$$C_{IJ} = \lambda(\underline{e}_I) \lambda(\underline{e}_J) \sin \gamma(\underline{e}_I, \underline{e}_J)$$

diagonal comp. are squared stretches in coordinate directions

off-diagonal components are related to shear between associated coord. dir.



Diagonal components:

$$\lambda(\underline{\hat{y}}) = \sqrt{\underline{\hat{y}} \cdot \underline{\hat{y}}}$$

$$\underline{A} = \{\underline{e}_i\} \quad A_{ij} = \underline{e}_i \cdot \underline{e}_j$$

$$\Rightarrow C_{II} = \underline{e}_I \cdot \underline{e}_I = \lambda^2(\underline{e}_I)$$

Off diagonal components:

$$\cos \theta(\underline{e}_I, \underline{e}_J) = \frac{\underline{e}_I \cdot \underline{e}_J}{\lambda(\underline{e}_I) \lambda(\underline{e}_J)} \quad C_{IJ} = \underline{e}_I \cdot \underline{e}_J$$

$$C_{IJ} = \lambda(\underline{e}_I) \lambda(\underline{e}_J) \cos \theta(\underline{e}_I, \underline{e}_J)$$

shear between two basis vectors

$$\gamma(\underline{e}_I, \underline{e}_J) = \underbrace{\theta(\underline{e}_I, \underline{e}_J)}_{\frac{\pi}{2}} - \theta(\underline{e}_I, \underline{e}_J)$$

$$\theta(\underline{e}_I; \underline{e}_J) = \frac{\pi}{2} - \gamma(\underline{e}_I, \underline{e}_J)$$

$$C_{IJ} = \lambda(\underline{e}_I) \lambda(\underline{e}_J) \underbrace{\cos\left(\frac{\pi}{2} - \gamma(\underline{e}_I, \underline{e}_J)\right)}_{\sin(\gamma(\underline{e}_I, \underline{e}_J))}$$

$\Rightarrow$  Components of  $\underline{\underline{C}}$  directly quantify stretch and shear unlike components of  $\underline{\underline{F}}$ .