

Lecture 10: Mohr Circle

- Logistics:
- HW 4 will update resubmissions soon
 - HW 2 now have 6/7 thank you
 - HW 3 4/7 please submit
 - HW 4 posted later today

Last time: Extremal Stress values

⇒ constrained optimization problem

- normal stresses ⇒ $(\underline{\sigma} - \lambda \underline{I}) \underline{v} = \underline{0}$

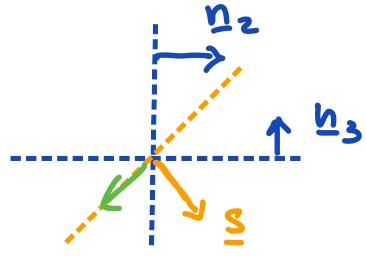
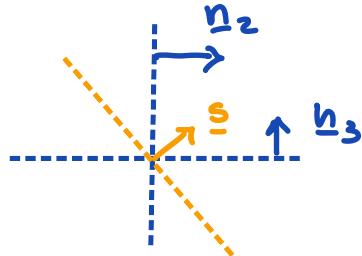
eigenvalues $\lambda_p = \sigma_p$ extremal normal stresses

Note we assume: $\sigma_1 > \sigma_2 > \sigma_3 \geq 0$

\underline{v}_p are normals to associated planes

- shear stresses

$$\begin{aligned}\tau_{23} &= \frac{1}{2}(\sigma_2 - \sigma_3) \leq \frac{1}{\sqrt{2}}(\pm n_2 \pm n_3) \\ \tau &= \frac{1}{\sqrt{2}}(n_2 - n_3)\end{aligned}$$



Today:

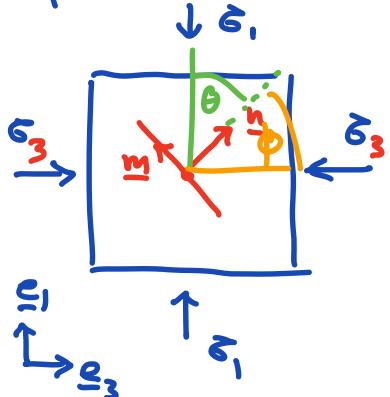
- Mohr circle

- Computing stress on fault

Mohr circle

graphical way to display normal & shear stresses on all planes.

2D case



θ is angle between $\underline{n} \cdot \underline{e}_1 = \cos \theta$

ϕ is angle between $\underline{n} \cdot \underline{e}_3 = \cos \phi$

$$\phi + \theta = \frac{\pi}{2} \quad \phi = \frac{\pi}{2} - \theta$$

$$\underline{n} = n_1 \underline{e}_1 + n_3 \underline{e}_3 \quad n_2 = 0$$

$$n_1 = \underline{n} \cdot \underline{e}_1 = \cos \theta$$

$$n_3 = \underline{n} \cdot \underline{e}_3 = \cos \phi = \cos\left(\frac{\pi}{2} - \theta\right) = \sin \theta$$

$$\Rightarrow \underline{n} = \begin{pmatrix} \cos \theta \\ \sin \theta \end{pmatrix}$$

$$\underline{m} = \begin{pmatrix} -\sin \theta \\ \cos \theta \end{pmatrix}$$

Stress in principal frame:

$$\underline{\underline{\sigma}} = \sigma_1 \underline{\underline{e}}_1 \otimes \underline{\underline{e}}_1 + \sigma_2 \underline{\underline{e}}_2 \otimes \underline{\underline{e}}_2 + \sigma_3 \underline{\underline{e}}_3 \otimes \underline{\underline{e}}_3$$

traction: $\underline{\underline{\sigma}} \underline{n} = \underline{\underline{\sigma}} \underline{n} = \sigma_1 \cos \theta \underline{\underline{e}}_1 + \sigma_3 \sin \theta \underline{\underline{e}}_3$

normal stress: $\sigma_n = \underline{n} \cdot \underline{t}_n = \sigma_1 \cos^2 \theta + \sigma_3 \sin^2 \theta$

use $\cos^2 \theta = \frac{1 + \cos 2\theta}{2}$

$\sin^2 \theta = \frac{1 - \cos 2\theta}{2}$

$$\Rightarrow \boxed{\sigma_n = \frac{\sigma_1 + \sigma_3}{2} + \frac{\sigma_1 - \sigma_3}{2} \underline{\cos 2\theta}}$$

shear stress: $\tau = \underline{m} \cdot \underline{t}_n = (\sigma_1 - \sigma_3) \sin \theta \cos \theta$

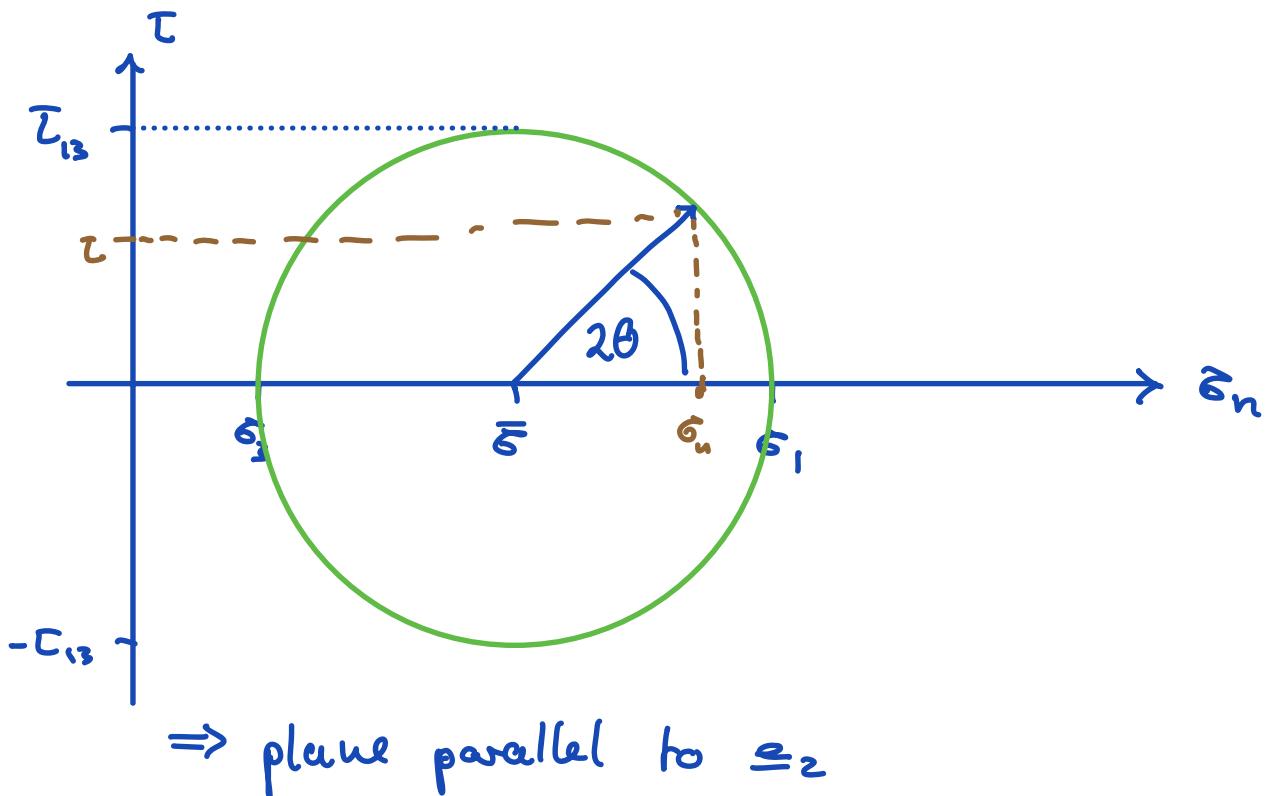
use $2 \sin \theta \cos \theta = \sin(2\theta)$

$$\Rightarrow \boxed{\tau = \frac{\sigma_1 - \sigma_3}{2} \underline{\sin 2\theta}}$$

Together these equations for a circle

in $\tau \sigma_n$ -space with radius $R = \frac{\sigma_1 - \sigma_3}{2}$

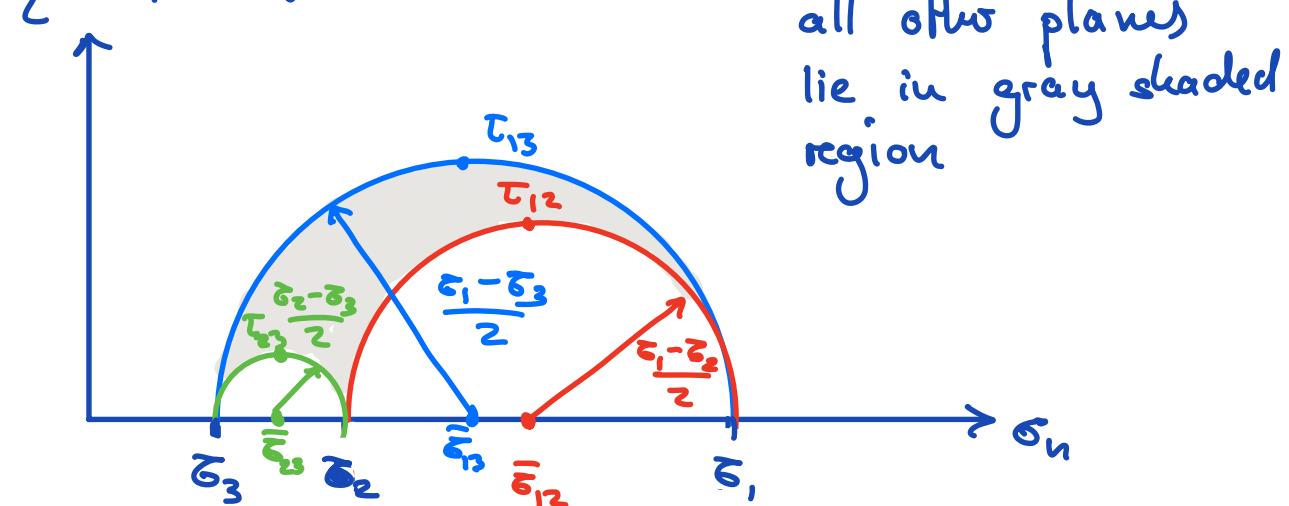
and center $(\frac{\sigma_1 + \sigma_3}{2}, 0)$

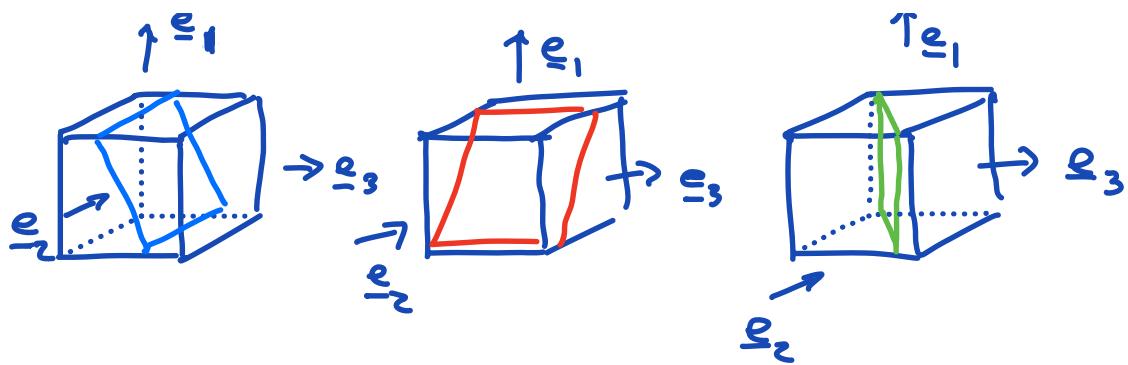


Mohr circles in 3D

Repeat argument for planes parallel to $\underline{\sigma}_1$ & $\underline{\sigma}_3$,

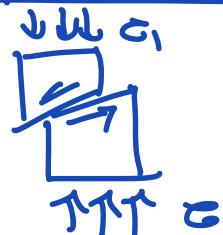
\Rightarrow eqns for 2 circles





Failure criteria for shear fracture

most common type of
brittle failure



Empirical criteria for shear failure

I, Tresca criterion

Fracture occurs when max shear stress

$\tau_{\max} = \tau_{13}$ reaches the shear strength σ_y

$$|\tau_{\max}| = \frac{\sigma_1 - \sigma_3}{2} = \sigma_y$$



not affected by

σ mean stress or intermediate stress

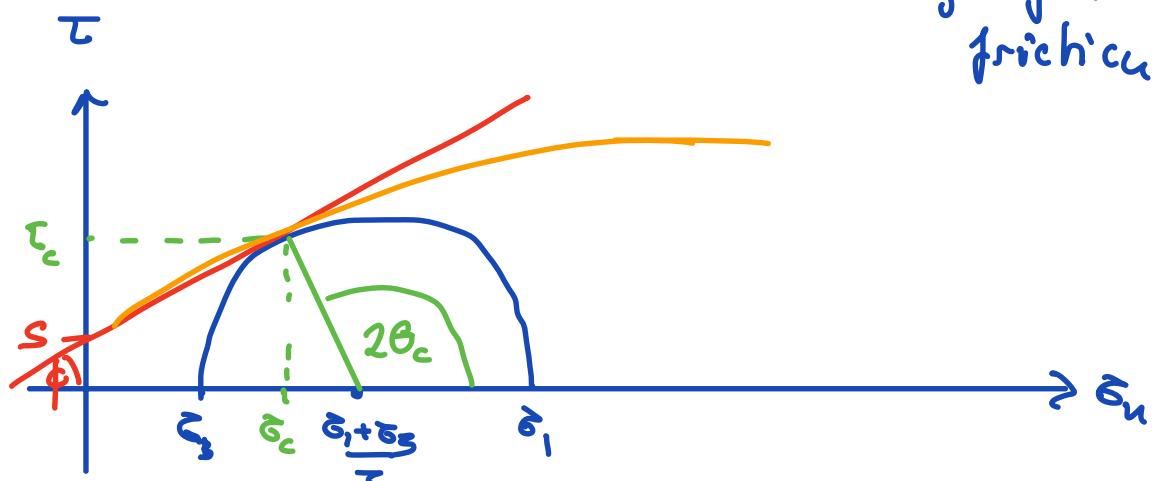
II Coulomb criterion

Fracture depends on both mag. of shear and normal stress

$$|\tau| = s + \mu' \sigma_n$$

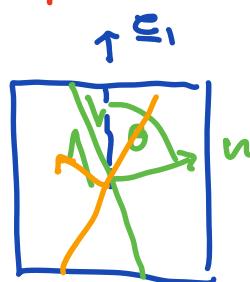
s = cohesive strength

$\mu' = \tan \phi \approx 0.6$ internal friction angle of internal friction $\phi \approx 30^\circ$



failure occurs $\tau_c < \tau_{\max}$

$$\theta_c \approx 60^\circ$$



Byerlee's law

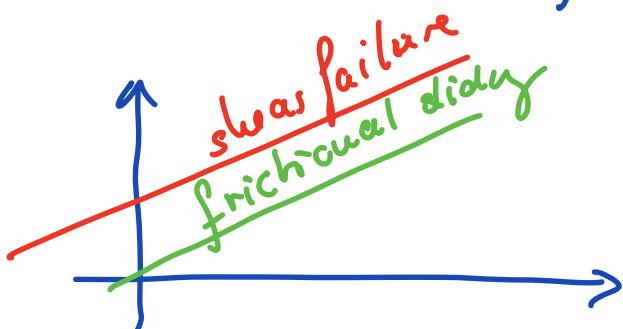
criterion for frictional sliding on fault

$$|\tau| = S_0 + \mu_0 \sigma_u$$

S_0 = cohesion of fault $\sim 1-10\text{ MPa}$

μ_0 = coefficient of friction

$\sim 0.5-0.8$



Strength of brittle rocks is determined
by frictional sliding