

Lecture 6: More on stress

check points on canvas

Logistics: - HW 1 (5/7) please submit

- HW 2 will be posted

⇒ make use of office hours if HW is not clear

Last time: - infinitesimal force balance

$$f = \gamma \rho \quad f = -\gamma, -\gamma$$

$$\frac{1}{A} \oint_{\partial \Omega} \underline{t}_n dA = \underline{0} \quad (\text{Vol. terms vanish})$$

- Cauchy's postulate

$$\underline{t}_n = \underline{t}(\underline{n}, \underline{x})$$

- 3rd law (Action-Reaction)

$$\underline{t}_n(-\underline{n}, \underline{x}) = -\underline{t}(\underline{n}, \underline{x})$$

- Cauchy's theorem

$$\underline{t}_n = \underline{\underline{\sigma}}(\underline{x}) \underline{n}$$

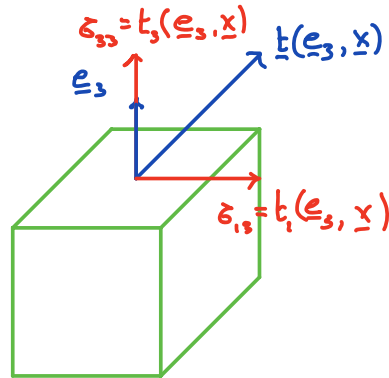
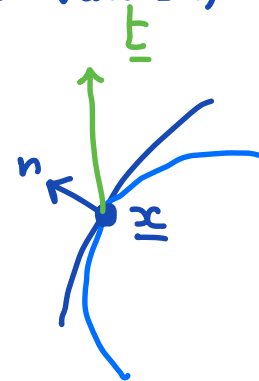
$\underline{\underline{\sigma}}$ = Cauchy stress

$$\underline{\underline{\sigma}} = \sigma_{ij} \underline{e}_i \otimes \underline{e}_j$$

$$\sigma_{ij} = \underline{t}_i(\underline{e}_j, \underline{x})$$

σ_{ij} = i-th component

of traction on jth coordinate plane

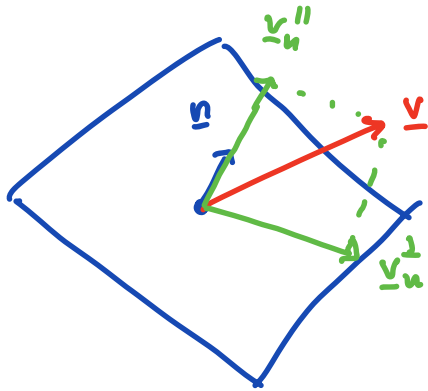


Today: Projections → shear & normal stress

Simple stress states

spherical + deviatoric stress

Projection Tensors



$$\underline{v} = \underline{v}_n'' + \underline{v}_n^\perp$$

dot product

$$\underline{v}_n'' = (\underline{n} \cdot \underline{v}) \underline{n}$$

$$\underline{v}_n^\perp = \underline{v} - \underline{v}_n''$$

$$\underline{v}_n'' = \underline{P}_n'' \underline{v}$$

$$\underline{v}_n^\perp = \underline{P}_n^\perp \underline{v}$$

\underline{P}_n'' & \underline{P}_n^\perp projection tensors

Use dyadic product: $(\underline{a} \otimes \underline{b}) \underline{c} = (\underline{b} \cdot \underline{c}) \underline{a}$
 $(\underline{n} \otimes \underline{n}) \underline{v} = \underline{n} (\underline{n} \cdot \underline{v})$

$$\underline{v}_n'' = (\underline{n} \otimes \underline{n}) \underline{v} = \underline{P}_n'' \underline{v}$$

$$\begin{aligned} \underline{v}_n^\perp &= \underline{v} - \underline{v}_n'' = \underline{I} \underline{v} - (\underline{n} \otimes \underline{n}) \underline{v} = (\underline{I} - \underline{n} \otimes \underline{n}) \underline{v} \\ &= \underline{P}_n^\perp \underline{v} \end{aligned}$$

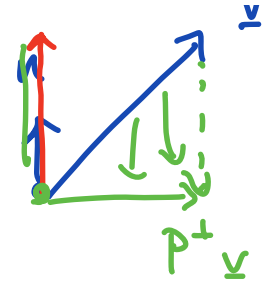
Projection tensors

$$\underline{P}_n'' = \underline{n} \otimes \underline{n}$$

$$\underline{P}_n^\perp = \underline{I} - \underline{n} \otimes \underline{n}$$

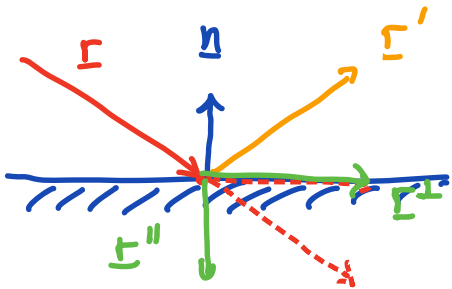
Properties:

$$\begin{aligned}
 \underline{\underline{P}} &= \underline{\underline{P}}^2 \\
 \underline{\underline{P}}^T &= \underline{\underline{P}} \\
 \underline{\underline{P}} + \underline{\underline{P}}^T &= \underline{\underline{I}} \\
 \underline{\underline{P}} - \underline{\underline{P}}^T &= \underline{\underline{0}}
 \end{aligned}$$



$$P^T v = P^T v$$

Reflections

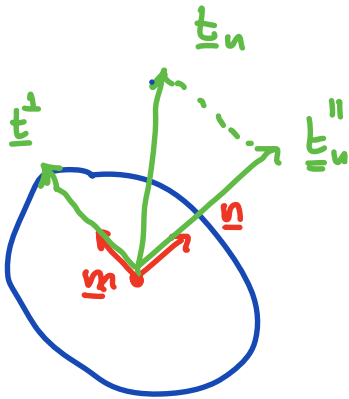


Fix notes

$$\begin{aligned}
 \underline{\underline{r}}' &= -\underline{\underline{r}}'' + \underline{\underline{r}}^T \\
 &= \underline{\underline{r}}^T - \underline{\underline{r}}'' \\
 &= \underline{\underline{P}}^T \underline{\underline{r}} - \underline{\underline{P}}'' \underline{\underline{r}} \\
 &= (\underline{\underline{P}}^T - \underline{\underline{P}}'') \underline{\underline{r}} \\
 &= (\underline{\underline{I}} - \underline{\underline{n}} \underline{\underline{n}}^T - \underline{\underline{n}} \underline{\underline{n}}^T) \underline{\underline{r}} \\
 &= (\underline{\underline{I}} - 2 \underline{\underline{n}} \underline{\underline{n}}^T) \underline{\underline{r}} \\
 &= \underline{\underline{R}} \underline{\underline{r}}
 \end{aligned}$$

Reflection tensor: $\underline{\underline{R}} = \underline{\underline{I}} - 2 \underline{\underline{n}} \underline{\underline{n}}^T$

Normal and Shear Stresses



$$\underline{t}_n = \underline{\sigma} \underline{n}$$

$$\underline{P}^n = \underline{n} \otimes \underline{n}$$

$$\underline{P}^t = \underline{I} - \underline{n} \otimes \underline{n} = \underline{m} \otimes \underline{m}$$

normal stress: $\underline{t}_n^n = \underline{P}^n \underline{t}_n = (\underline{n} \otimes \underline{n}) \underline{t}_n$

$$= (\underline{n} \cdot \underline{t}_n) \underline{n} = \sigma_n \underline{n}$$

σ_n is mag. of normal stress

$$\sigma_n = \underline{n} \cdot \underline{t}_n = \underline{n} \cdot \underline{\sigma} \underline{n}$$

$$\sigma_n = n_i \sigma_{ij} n_j$$

shear stress: $\underline{t}_n^t = \underline{P}^t \underline{t}_n = (\underline{m} \otimes \underline{m}) \underline{t}_n = \underbrace{(\underline{m} \cdot \underline{t}_n)}_{\tau} \underline{m}$

mag. of shear stress: $\tau = \underline{m} \cdot \underline{t}_n = \underline{m} \cdot \underline{\sigma} \underline{n} = m_i \sigma_{ij} n_j$

$\sigma_n > 0$ tensile stress

$\sigma_n < 0$ compressive stress

From geometry: $|\underline{t}_n|^2 = \sigma_n^2 + \tau^2$

Simple states of stress

I) Hydrostatic stress

$$\underline{\underline{\sigma}} = -p \underline{\underline{I}} = \begin{bmatrix} -p & 0 & 0 \\ 0 & -p & 0 \\ 0 & 0 & -p \end{bmatrix} \quad \underline{t} = -p \underline{n}$$

$$\underline{t}_n = \underline{\underline{\sigma}} \underline{n} = (-p \underline{\underline{I}}) \underline{n} = -p \underline{n}$$

$$\begin{aligned} \underline{t}'' &= p'' \underline{t}_n = (\underline{n} \otimes \underline{n}) (-p \underline{n}) = -p (\underline{n} \otimes \underline{n}) \underline{n} \\ &= (\cancel{\underline{n}} \cdot \underline{n}) \underline{n} (-p) = -p \underline{n} \end{aligned}$$

$$\underline{t}'' = \underline{t}_n \quad \Rightarrow \quad \underline{t}^\perp = 0$$

$$\left. \begin{array}{l} \text{normal stress: } \underline{\sigma}_n = -p \\ \text{shear stress: } \tau = 0 \end{array} \right\} \text{ on all planes}$$

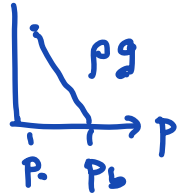
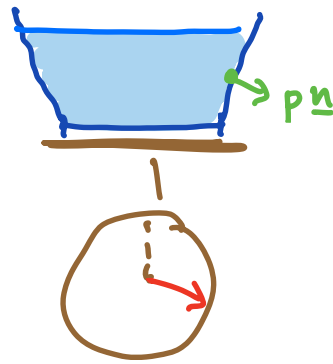
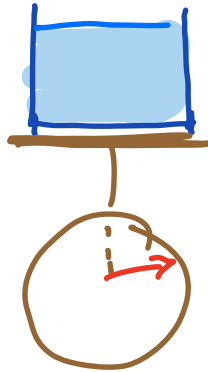
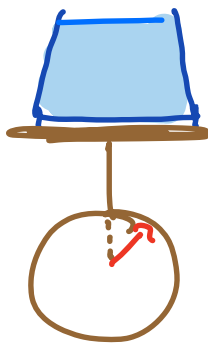
Pascal's law:

The pressure in a fluid at rest is independent of the direction of a surface.

Hydrostatic paradox:

force on base is same

$$F = A p$$



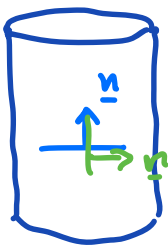
II Uniaxial stress

$$\underline{\underline{\underline{\underline{\sigma}}}} = \sigma \underline{\underline{\underline{\underline{\gamma}}}} \otimes \underline{\underline{\underline{\underline{\gamma}}}}$$

$\underline{\underline{\underline{\underline{\gamma}}}}$ = unit vector

$$\underline{t}_n = \underline{\underline{\underline{\underline{\sigma}}}} \underline{n} = \sigma (\underline{\underline{\underline{\underline{\gamma}}}} \otimes \underline{\underline{\underline{\underline{\gamma}}}}) \underline{n} = \sigma (\underline{\underline{\underline{\underline{\gamma}}}} \cdot \underline{n}) \underline{\underline{\underline{\underline{\gamma}}}}$$

$$\underline{\underline{\underline{\underline{\gamma}}}} \downarrow \underline{t}_n = \sigma (\underline{\underline{\underline{\underline{\gamma}}}} \cdot \underline{n}) \underline{\underline{\underline{\underline{\gamma}}}}$$



$$\underline{t}_n = \pm \sigma \underline{\underline{\underline{\underline{\gamma}}}}$$

$$\underline{t}_n = \sigma (\underline{\underline{\underline{\underline{\gamma}}}} \cdot \underline{n}) \underline{n} = 0$$

traction is always parallel to $\underline{\underline{\underline{\underline{\gamma}}}}$

III Pure shear stress

two directions \underline{x} and \underline{y} perpendicular $\underline{x} \cdot \underline{y} = 0$

$$\underline{\underline{\sigma}} = \tau (\underline{x} \otimes \underline{y} + \underline{y} \otimes \underline{x})$$

$$\underline{t}_n = \underline{\underline{\sigma}} \underline{n} = \tau [(\underline{x} \otimes \underline{y}) \underline{n} + (\underline{y} \otimes \underline{x}) \underline{n}]$$

$$= \tau [(\underline{y} \cdot \underline{n}) \underline{x} + (\underline{x} \cdot \underline{n}) \underline{y}]$$

$$\underline{n} = \underline{y}: \underline{t}_n = \tau [(\underline{y} \cdot \underline{y}) \underline{x} + (\underline{x} \cdot \underline{y}) \underline{y}]$$
$$= \tau \underline{x}$$

$$\underline{n} = \underline{x}: \underline{t}_n = \tau \underline{y}$$

IV Plane stress

If there exist a pair of orthog. vectors \underline{x} & \underline{y} s.t. matrix representation in frame $\{\underline{x}, \underline{y}, \underline{x} \times \underline{y}\}$ is of form

$$[\underline{\underline{\sigma}}] = \begin{pmatrix} \sigma_{11} & \sigma_{12} & 0 \\ \sigma_{21} & \sigma_{22} & 0 \\ 0 & 0 & 0 \end{pmatrix}$$

Q: Is uniaxial stress a plane stress?

$$\underline{\underline{\sigma}} = \sigma \underline{x} \otimes \underline{x}$$

pick a frame e

$$\{ \underline{x}, \underline{e}_2, \underline{e}_3 \}$$

$$\underline{x} \cdot \underline{e}_2 = 0 \quad \underline{x} \cdot \underline{e}_3 = 0$$

don't need to specify

other basis vectors

(not unique)

$$\underline{\underline{\sigma}} = \sigma_{ij} \underline{e}_i \otimes \underline{e}_j$$

$$\sigma_{ij} = \underline{e}_i \cdot \underline{\underline{\sigma}} \underline{e}_j$$

$$= \underline{e}_i \cdot (\sigma \underline{x} \otimes \underline{x}) \underline{e}_j$$

$$= \sigma \underline{e}_i \cdot (\underline{e}_1 \otimes \underline{e}_1) \underline{e}_j = \sigma \underline{e}_i \cdot (\underline{e}_1 \cdot \underline{e}_j) \underline{e}_1$$

$$\sigma_{ij} = \sigma (\underline{e}_1 \cdot \underline{e}_j) (\underline{e}_i \cdot \underline{e}_1)$$

$$\sigma_{11} = \sigma (\underline{e}_1 \cdot \underline{e}_1) (\underline{e}_1 \cdot \underline{e}_1) = \sigma$$

$$\sigma_{12} = \sigma (\underline{e}_1 \cdot \underline{e}_2) (\underline{e}_1 \cdot \underline{e}_1) = 0$$

$$\sigma_{21} = 0$$

$$[\underline{\underline{\sigma}}] = \begin{bmatrix} \sigma & c & 0 \\ e & 0 & 0 \\ c & c & 0 \end{bmatrix} \quad \checkmark \quad \text{plane stress}$$

Spherical & Deviatoric Stress tensors

$$\underline{\underline{\sigma}} = \underline{\underline{\sigma}}_s + \underline{\underline{\sigma}}_D$$

spherical stress tensor: $\underline{\underline{\sigma}}_s = -p \underline{\underline{I}} \quad p = -\frac{1}{3} \text{tr}(\underline{\underline{\sigma}})$

deviatoric stress tensor: $\underline{\underline{\sigma}}_D = \underline{\underline{\sigma}} - \underline{\underline{\sigma}}_s$
 $= \underline{\underline{\sigma}} + p \underline{\underline{I}}$

The pressure $p = -\frac{1}{3} \text{tr}(\underline{\underline{\sigma}})$ is mean normal traction.

Spherical stress \rightarrow volumetric changes

Deviatoric stress \rightarrow changed shape of body